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Article

Methodology for Virtual Prediction of Vehicle-Related Particle Emissions and Their Influence on Ambient PM₁₀ in an Urban Environment

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Abstract: As a result of rising environmental awareness, vehicle-related emissions such as particulate matter are subject to increasing criticism. The air pollution in urban areas is especially linked to health risks. The connection between vehicle-related particle emissions and ambient air quality is highly complex. Therefore, a methodology is presented to evaluate the influence of different vehicle-related sources such as exhaust particles, brake wear and tire and road wear particles (TRWP) on ambient particulate matter (PM). In a first step, particle measurements were conducted based on field trials with an instrumented vehicle to determine the main influence parameters for each emission source. Afterwards, a simplified approach for a qualitative prediction of vehicle-related particle emissions is derived. In a next step, a virtual inner-city scenario is set up. This includes a vehicle simulation environment for predicting the local emission hot spots as well as a computational fluid dynamics model (CFD) to account for particle dispersion in the environment. This methodology allows for the investigation of emissions pathways from the point of generation up to the point of their emission potential.

Keywords: particle emissions; particulate matter; non exhaust emissions; brake wear emissions; exhaust emissions; tire particle emissions; TRWP; tire wear; ambient air quality; PM inventory; emissions prediction



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1. Introduction

As a result of negative health effects linked to particulate matter exposure, limits were introduced for ambient PM₁₀ and PM_{2.5} concentrations in Europe. On the vehicle side, exhaust PN and PM emissions were limited as well. However, it is expected that non-exhaust sources already exceed exhaust emissions. Brake wear and TRWP are not subject to legal regulation [1]. The relationship between ambient PM concentrations and vehicle emissions is very complex and influenced by numerous parameters. In order to develop effective protective measures, a holistic approach is necessary. The main sources have to be identified and put into relation. Furthermore, the particle dispersion in the environment has to be taken into account. The exposure concentration level of each particle sources has to be determined individually. Within this work, a novel simulation approach is presented to predict vehicle-related emissions as well as particle dispersion in the environment.

In a first step, a literature review was conducted regarding methods for vehicle-related particle emission prediction. Domingues et al. used an artificial neuronal network (ANN) as well as symbolic regression in order to predict exhaust emissions such as O₂, NO_x and particle number [2]. Ghiasi et al. applied an ANN for the prediction of parameters for an exhaust after-treatment system [3]. Ricciardi presented a semi-empirical brake

wear model. Furthermore, brake wear emissions were predicted using an ANN [4]. Riva et al. used an FEA approach for the prediction of macroscopic brake wear and air-borne particle emissions. Therefore, the influence of the local contact pressure and sliding speed was included into the model. For this purpose, wear and emission maps from pin-on-disc tests were implemented [5]. Sasa et al. presented an approach for the prediction of PM₁₀ and PM_{2.5} brake wear emissions using a neural network. Based on an inertia brake dynamometer, the neural network was trained and validated. In addition, different ANN algorithms were compared [6]. Nguyen et al. developed a mathematical tire wear model. The model includes the influence of directional effects and wear history. Additionally, local contact conditions such as temperature, sliding speed and contact pressure were implemented. The model was validated by experiment (Grosch wheel) [7]. Braghin et al. presented a methodology for the prediction of global tire wear and local wear distribution. A mathematical tire model was combined with an experimentally determined local friction and wear law [8]. Ivanov modelled the influence of longitudinal and lateral slip on tire wear. The model was applied for the investigation of wear loss [9]. Feißel et al. presented an approach for the estimation of air-borne TRWP emissions as a function of lateral and longitudinal acceleration as well as vehicle velocity [10]. As it can be seen from this review, several approaches for modeling and prediction of exhaust, brake and tire emissions already exist. However, few of them include air-borne particles.

In order to improve the evaluation of the toxic potential of air-borne particles, it is necessary to create a holistic methodology that can be used to determine the exposure concentration in the ambient air. Therefore, a CFD model can provide insights into the particle dispersion process. Likewise, a literature review has been conducted for this topic. Wingstedt and Reif analyzed the particle transport and deposition for different wind conditions in an urban environment. The study investigated a vehicle as a moving emission source. It was found that the wind velocity has a decisive influence on the horizontal and vertical particle dispersion [11]. Nikolova et al. investigated the dispersion of ultrafine particles (UFP) based on an inner-city scenario in Antwerp [12]. Gidhagen et al. analyzed the particle concentration within a street canyon in order to determine the relevance of aerosol dynamic processes such as coagulation and deposition [13]. Amorim et al. investigated the influence of urban trees on the dispersion of toxicity in the air in Aveiro and Lisbon. Therefore, the respective mean wind values were applied as boundary conditions. Due to the aerodynamic blocking effect of the trees, a reduced exchange rate of the polluted air was observed [14]. Wang and Zhang analyzed the air quality near streets in Los Angeles. Since an increase in local turbulence due to the traffic was expected, simplified block-shaped modelled vehicles were included in the model [15]. Pospisil and Jicha investigated the relationship between the wind speed and the associated particle resuspension. The study showed that above a certain threshold of the wind speed, deposited particles are resuspended, which leads to an increased particle concentration within street canyons [16]. Camelli et al. analyzed the influence of tall buildings (chimney effect) on the vertical particle dispersion in Manhattan. Furthermore, the influence of the street layout on the lateral dispersion was investigated [17].

As shown in the review, various individual approaches exist for the prediction of emissions as a function of vehicle dynamic parameters. The same applies for the prediction of particle dispersion within street canyons. However, a holistic approach connecting both models is still missing.

Within this work, a novel methodology for the tracking of particle pathways from the point of formation up to the dispersion in the environment is presented. It contains a first approach for the generation of an emission database, the derivation of emission predictors and finally the transfer into a flow model to account for particle-air-interaction. At first, an emission database for all relevant particle sources was created based on field trails. In a next step, a methodology for emission prediction was derived and implemented into a vehicle simulation environment. To analyze the influence on the ambient air quality, a corresponding CFD model was applied. Consequently, particle dilution and trans-location

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2. Materials and Methods

2.1. Framework

Figure 1 shows the framework of a simulation environment for virtual emission prediction and tracking in a first step, an emission database is created based on test vehicle trials. Thereafter, the most relevant brake and TRWP emissions are measured with mass measurement systems. Additionally, vehicle parameters (ranger, speed, lateral speed and longitudinal and lateral acceleration) are recorded. Based on the database, a methodology for a qualitative prediction of particle emissions is derived. In a next step, a simulation of a flow model is derived based on the existing data of the example (City of Erfurt, Germany) in this paper. Within the framework, the measured data of the test vehicles and the resulting ambient air concentrations are calculated for each other. Furthermore, an emission prediction model is proposed as an interface of the two further models to enable further analyses on emission hotspots. Finally, the predicted particle mass flow is transferred into a corresponding CFD model. Based on the model, the ambient air concentration as a function of wind properties (magnitude and direction) and particle size is estimated. The objective of this methodology is to determine the correlation between vehicle-related emissions and ambient air quality.

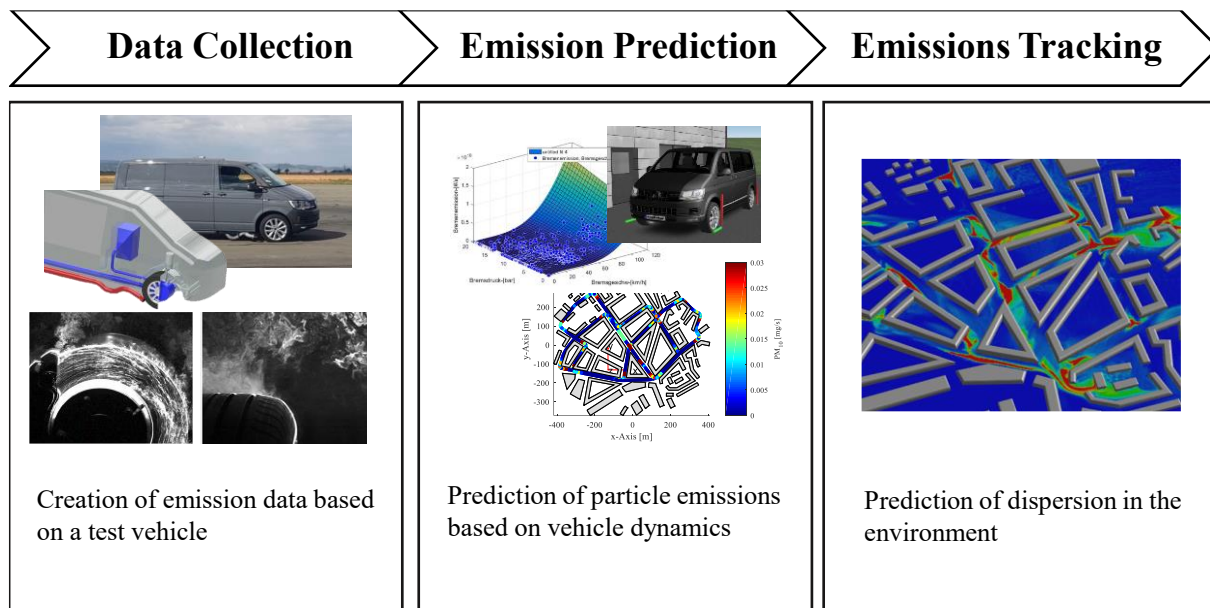


Figure 1. Methodology for particle measurement (creation of a database), virtual emission prediction and particle tracking within the environment.

2.2. Measurement Set-Up

The measurements were performed based on a test vehicle under real road conditions. Consequently, brake TRWP, TRWP and exhaust emissions were measured for an vehicle under realistic conditions. The vehicle, shown in Figure 2, has a diesel engine with a displacement of 1968 cm³ and a maximum power of 75 kW at 3500 rpm. The total mass of the vehicle is 2150 kg. It is equipped with summer tires (255/45 R18) and ECE brake pads. Exhaust emissions are sampled by a probe placed inside the exhaust pipe. An exhaust flow meter (EFM) is used to determine the exhaust gas volume flow. Brake wear and TRWP emissions are sampled by two separate constant volume sampling systems (CVS). Subject

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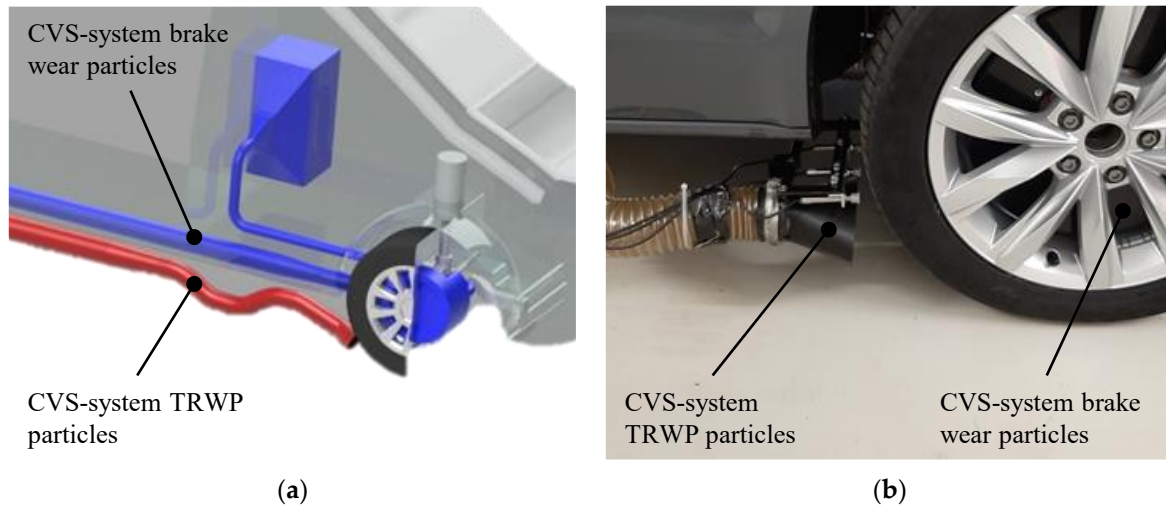


Figure 2. Test setup for vehicle-based particle emission measurement: (a) schematic illustration of sampling systems for brake wear (blue) and TRWP particles (red)—(b) measurement setup mounted to the test vehicle [18].

In order to collect the maximum amount of particles, the brake system is fully enclosed and separated from ambient air. An electric fan generates an air flow through the system to evacuate the particles and transport them into a measurement tunnel. On the inlet side a filter stage (class H13) is integrated to avoid background influences. The measurement setup for TRWP emissions consists of a sampling funnel, a flexible hose and a filter on a further measurement chamber. In combination with the CVS system for brake emissions, the disturbance of the TRWP measurement by brake particles is avoided. The TRWP funnel is mounted to the wheel carrier. This design allows for obtaining a measurement to the perpendicular to the tread surface even during cornering. However, due to inlet losses due to inlet efficiency and particle deposition as a function of vehicle speed and particle size have to be taken into account [18].

Since it is the aim of this study to investigate the correlation between vehicle-related emissions and ambient air quality, it was focused on RMM concentration. Therefore, a TSI Dusttrak 8530 sensor resolution was used with an additional RML PM₁₀ separator. The system is applicable for the measurement of the particle mass concentration (mg/m³) in a size range from 100 nm to 10 µm. Furthermore, the particle number concentration (#/cm³) was used for these purposes, purposes as a particle counter (TSI) with a range of measuring range from 0.2 µm up to 10 µm. But as stated both systems provide only measurements with a sample flow rate. Since exhaust emissions are mainly in a size range below 100 nm, exhaust PM was calculated based on the number concentration assuming a mean particle diameter of 100 nm and a density of 1/g/cm³.

$$PC = ME \cdot PC_s \cdot \dot{V}_s \tag{1}$$

In order to compare different particle sources, the sample volume flow must be taken into account. While the brake wear and TRWP particles are sampled with an approximately constant volume flow (120 m³/h), the exhaust gas volume flow varies depending on the load condition and the engine rpm. The respective volume flows are recorded with volume flow sensors. By multiplying particle concentration PC and sample flow \dot{V}_s , the corresponding normalized emission factor $\dot{P}C$ ([mg/s] and [# /s]) can be determined. Furthermore, a measurement efficiency factor ME is introduced accounting for particle losses due to inlet and deposition losses.

To investigate the relationship between vehicle-related emissions and ambient PM₁₀, an inner-city scenario based on the example of the city of Erfurt was chosen. The focus is on a specific street (Bergstraße), where an environmental measuring station for air pollutants monitoring is located. This allows the comparison of measured and predicted ambient air concentrations.

As shown in Figure 3a, a vehicle simulation model of the district was set up with the use of the commercial vehicle simulation software IPG CarMaker. The orientation of the streets including all traffic lights, traffic signs, lanes, lane markings and parking bays was modeled. A vehicle model is selected to emulate the measurement vehicle, which was used for the generation of the emission data base. Based on the model, a virtual test cycle within the city environment is conducted. By implementing emission predictors, the local emission contribution can be estimated.

Furthermore, the particle dispersion in the environment is investigated by setting up a corresponding CFD model of the city scenario. Therefore, the commercial CFD code ANSYS Fluent was used. For simplification, only the buildings were represented. Details such as cars and trees are neglected. Based on the model shown in Figure 3b, the local wind vectors within the street canyons can be calculated as a function of the global wind speed and global wind direction. The model contains the area within a circumference of 1 km of the environmental measurement station (Bergstraße) and has a height of 200 m.

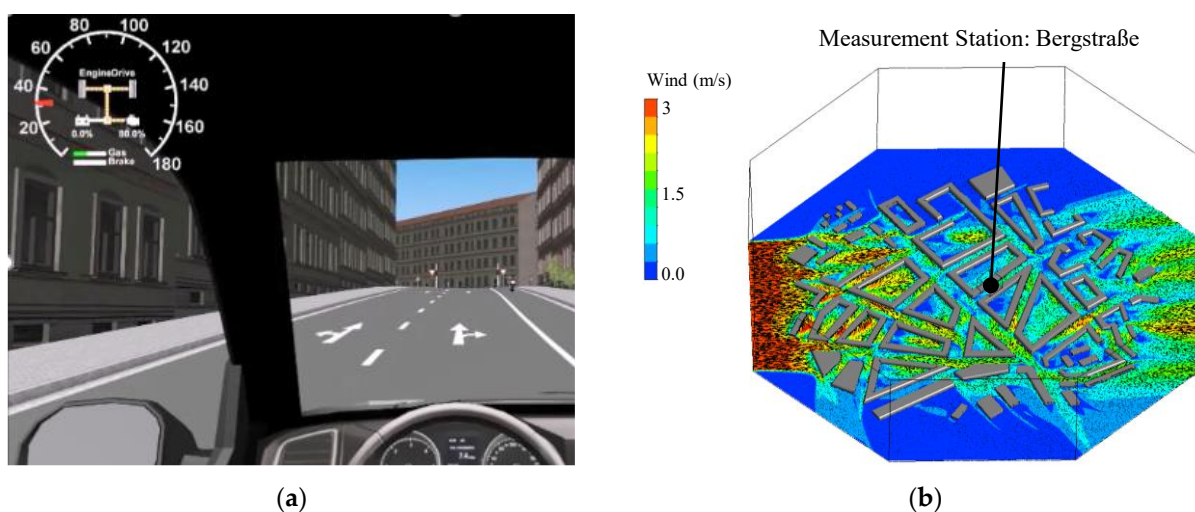


Figure 3. Simulation environment (Erfurt, Germany): (a) vehicle simulation model (b) CFD model (global wind direction: west/global wind speed 10 km/h).

The model covers the particle dispersion in the environment. A hybrid-meshing algorithm was applied resulting in a total cell count of 1.5×10^6 . The boundary region of the CFD was ANSYS Fluent 10 prism shaped inflation layers with a first layer thickness of 1×10^{-3} m to ensure a y^+ value of $y^+ = 1$. The 8 edge surfaces of the model volume can be individually defined as velocity inlets to account for different wind directions. In case the surface is not assigned as velocity inlet it is defined as a pressure outlet. For the flow calculation, the k- ω -SST turbulence model is applied. A turbulent intensity of 5% and a turbulent viscosity ratio of 10 is assumed for both inlets and outlets.

To predict the particle dispersion in the environment, a method for the implementation of the previously predicted particle emissions was developed. For this purpose, an input file is required. This file contains the normalized emission factor PC , the injection coordinate position (x, y, z) , the particle diameter d_p and the vehicle velocity vector \vec{v}_v . The particles are injected after the flow field calculation reached its convergence criterion (residuals $< 1 \times 10^{-3}$). The particle tracking is based on the discrete phase model (DPM). Therefore, spherical model particles (parcels) in a size range between 1 and 10 μm are

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3. Results

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3.1. Influence Parameters of Particle Generation

For the investigation of the correlation between vehicle dynamics and particle emissions, field trials were conducted. For all vehicle-related particle sources, the normalized emission factor was determined as a function of vehicle dynamic parameters. The study is based on a real driving emissions-compliant (RDE) driving cycle (EU 2018/1832). The cycle is composed of city segments ($\leq 60 \text{ km/h}$), rural segments ($60\text{--}90 \text{ km/h}$) and motorway segments ($90\text{--}145 \text{ km/h}$) with a distance share of 33% each and a total length of 87 km. The average cycle speed is 55 km/h.

Internal combustion engines convert chemical energy into mechanical energy and heat. Under ideal conditions, only carbon dioxide, water and nitrogen are generated. Under real conditions, the interruption of the reaction chains results in undesirable products. These include carbon monoxide, unburned and partially burned hydrocarbons and oxides of nitrogen as well as solids in the form of particle emissions [19].

In Figure 4 the correlation between the driving conditions and exhaust emissions is shown. As expected, the highest emission values are observed during acceleration events in the upper engine torque and rpm-range. The vehicle complies with the Euro6 emissions standard and is equipped with a diesel particulate filter. The measurement results confirm the effectiveness of the exhaust aftertreatment system. Overall, the measured emission values are at a comparatively low level.

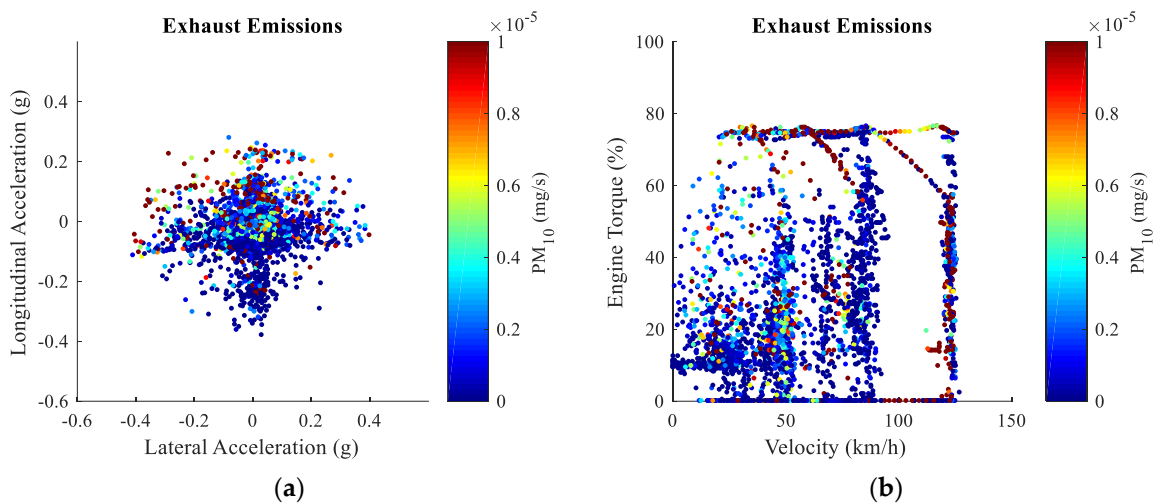


Figure 4. Correlation between vehicle dynamics and particle generation: exhaust emissions as a function of (a) vehicle acceleration, (b) engine torque and velocity.

Friction brakes are based on the conversion of kinetic energy into thermal energy. They are most commonly used to control and regulate the vehicle speed of vehicles with conventional internal combustion engines. One way to reduce brake-related emissions effectively is to use vehicles with regenerative braking systems (electric drive or hybrid drive). Under specific operating conditions, the regenerative electric drive can replace the function of the friction brake [20]. The brake system consists of the brake disk and the brake linings. During the braking event, the boundary layer undergoes continuous buildup and degradation due to wear mechanisms. The third layer is composed of the

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In Figure 5 the correlations between deceleration, vehicle speed, brake pressure and brake wear emissions are shown. The measurement was carried out based on an RDE-compliant driving cycle. As expected, brake wear emissions occur during deceleration events. The fact that virtually no emissions emerge in a low deceleration range (0–0.06 g) can be attributed to the effect of driving resistance forces (i.e., air resistance, roll resistance, transmission resistance). Furthermore, a dependency from the brake pressure can be derived. However, the vehicle speed shows the highest influence. Braking maneuvers with low brake pressures at high-speed result in accordingly high particle emission values.

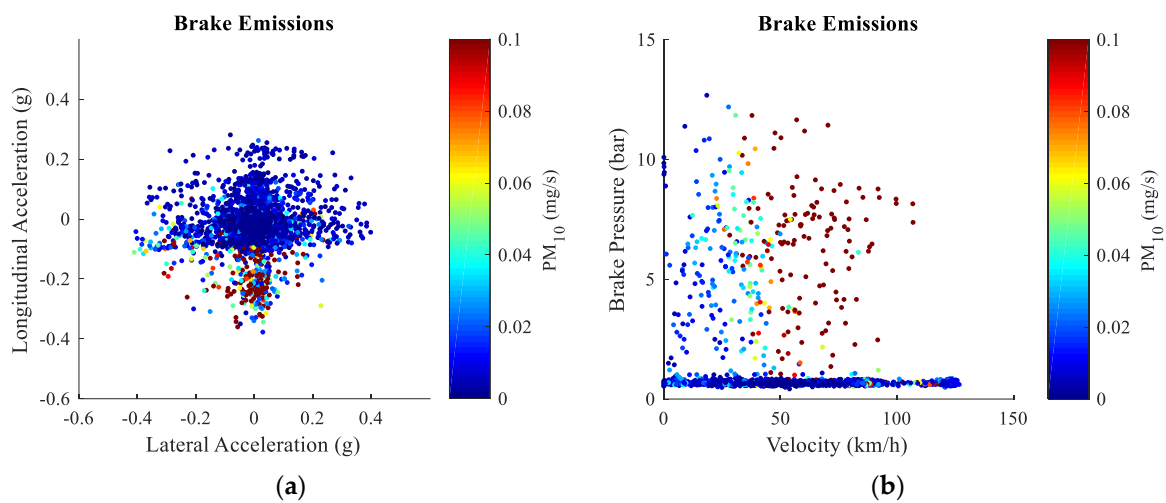


Figure 5: Correlation between vehicle dynamics and particle generation: brake wear emissions as a function of (a) vehicle deceleration, (b) brake pressure and vehicle speed.

The transmission of longitudinal and lateral forces between the tires and the road leads to wear generation. TRWP particles are generated mainly by the shear forces between the tire and the road, which leads to mechanical abrasion and the formation of coarse particles (1–10 μm). In addition to the air-borne fraction, a large part of the abrasion particles (90%) is emitted in the form of plastic particles (size up to 5 μm). In the case of high tire temperatures, increased tire temperatures can lead to oxidative processes. Consequently, secondary particles (< 100 nm) can occur [1].

Figure 6 shows the relationship between the driving dynamics and TRWP emissions. Opposed to the measurement of exhaust and brake emissions, the driving tests were carried out on a proving ground to reduce background influences from other traffic participants. Parts of the conducted tests based on the RDE cycle (RDE urban cycle) did not show correlation between the driving dynamics and particle emissions due to the opening system configuration and the high number of disturbance variables in public traffic. Therefore, the main sequence of the RDE cycle was reproduced on the test track. However, due to the layout of the test site, the motorway segments could not be reproduced to full extent resulting in a shorter cycle distance of 37 km. TRWP emissions mainly occur during accelerated driving maneuvers as a result of the force transmission. The largest contribution of tire emissions is observed during cornering. Furthermore, the relative velocity between the tire and the road (slip velocity) is of importance.

$$PC = f(d_p, \rho_p, \dots) * P_{b,t,e} \tag{2}$$

strong simplification, vehicle emissions can be estimated as a function of power transmission. The frictional power, transmitted at the vehicle brakes and tires, is primarily converted into thermal energy. A further power component is responsible for the wear generation. Therefore, a proportionality between power transmission $P_{b,t,e}$ and the particle emission factor PC is assumed, which can be describe by a transfer function $f(d_p, \rho_p)$ taking into account particle properties such as diameter d_p and density ρ_p .

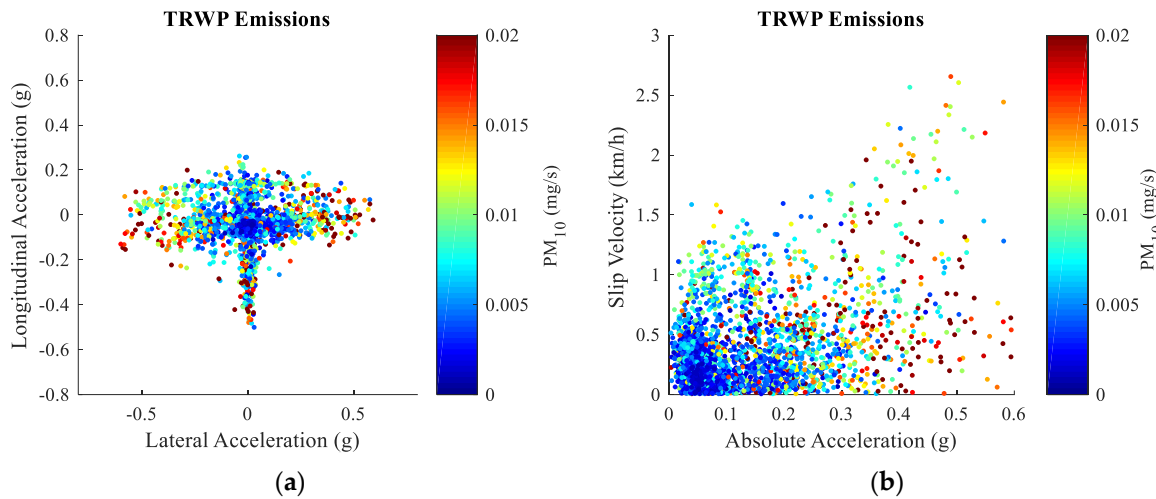


Figure 6. Correlation between vehicle dynamics and particle generation: TRWP emissions as a function of (a) longitudinal and lateral acceleration and (b) slip velocity.

The frictional power transmitted, during braking P_f is calculated based on the brake pressure and TRWP were evaluated. The P_f between a brake disc and pads, the coefficient of friction μ and the disc rotation speed ω . As shown in Figure 6, the brake pressure and the vehicle speed are two main influence parameters for brake particle emissions. The frictional power transmitted at the tire-road interface P_t is determined based on the longitudinal and lateral power direction as well as responsible for the wear generation. Therefore, the proportionality between power transmission $P_{b,t,e}$ and the particle emission factor PC is assumed, which can be described by a transfer function $f(d_p, \rho_p, \dots)$ taking into account particle properties such as diameter d_p and density ρ_p . The drag coefficient c_d and the vehicle cross sectional area A_v have to be taken into account. As shown in Figure 6, lateral and longitudinal acceleration are the main influence variables of TRWP formation. Similarly, exhaust emissions can be estimated as a function of engine power P_e taking into account engine torque M_e and engine rotational speed n_e (Figure 4). The frictional power transmitted at the tire-road interface P_t is determined based on the wheel forces acting on longitudinal and lateral direction as well as the total wheel velocity v_w as a function of the tire radius r and the slip velocity v_s . The main component of P_t is the frictional power P_f transmitted at the longitudinal brake and lateral direction. At high and low speed the respective PM_{10} and PN concentration of sampling losses σ are neglected drag coefficient c_d and the vehicle cross sectional area A_v have to be taken into account. As shown in Figure 6, lateral and longitudinal acceleration are the main influence variables of TRWP formation. Similarly, exhaust emissions can be estimated as a function of engine power P_e taking into account engine torque M_e and engine rotational speed n_e (Figure 4).

Table 1 outlines the respective emission factors for PM_{10} and PN as a function of power transmission while assuming a direct proportionality. Therefore, the frictional power transmitted at the front right brake and tire was calculated and related to the respective PM_{10} and PN concentration. Sampling losses are neglected in a first step, since the measurement efficiency factor (ME) cannot be validly determined. However, it has to be considered that especially the TRWP-measurement is subject to major sampling losses due to the open sampling system configuration [18].

Table 1. Estimation of vehicle-related particle emissions (PM₁₀ and PN) as a function of power transmission.

Source	Power (Kw)	PM ₁₀ (mg/kWs)	PN (#/kWs)
Brake	$P_b = p_b A_b \mu_b v_s$	~ 0.184	$\sim 1.15 * 10^9$
TRWP	$P_t = (m_{va_{xy}} + 0.5 \rho_L c_d A_v v_v^2) * v_s$	~ 0.051	$\sim 4.21 * 10^8$
Exhaust	$P_e = 2\pi M_e n_e$	$\sim 2.86 * 10^{-7}$	$\sim 3.97 * 10^5$

According to Table 1 the brake system shows the highest power related emission factors. TRWP emissions occur as the second biggest particle source while exhaust emissions do not play a significant role for a test vehicle complying with the Euro6 standard.

Even though the methodology is based on a strong simplification, it can be applied to a qualitative estimation of the respective particle contribution. Furthermore, it provides a framework, which can be optimized in the future.

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3.2. Prediction of Particle Generation

In a next step, the power-related emission factors were implemented into the vehicle simulation environment shown in Figure 3. The required vehicle parameters (i.e., velocity, longitudinal and lateral acceleration, slip velocity, brake pressure, engine torque and engine speed) were used as input variables. In the next step, a test cycle of 8 km length was conducted within the virtual test environment to investigate the local PM₁₀ contribution of each particle source.

Figure 7 demonstrates the predicted brake wear, TRWP and exhaust particle emissions (PM₁₀) along the vehicle trajectory. As expected, brake wear emission hot spots occur mainly in front of corners and traffic light intersections, where a reduction in speed or braking to a standstill is required. TRWP emissions are found in the braking area before and the acceleration area after corners and intersections. However, maximum values are observed during cornering. Exhaust emissions are found in areas where acceleration is required to reach the desired speed.

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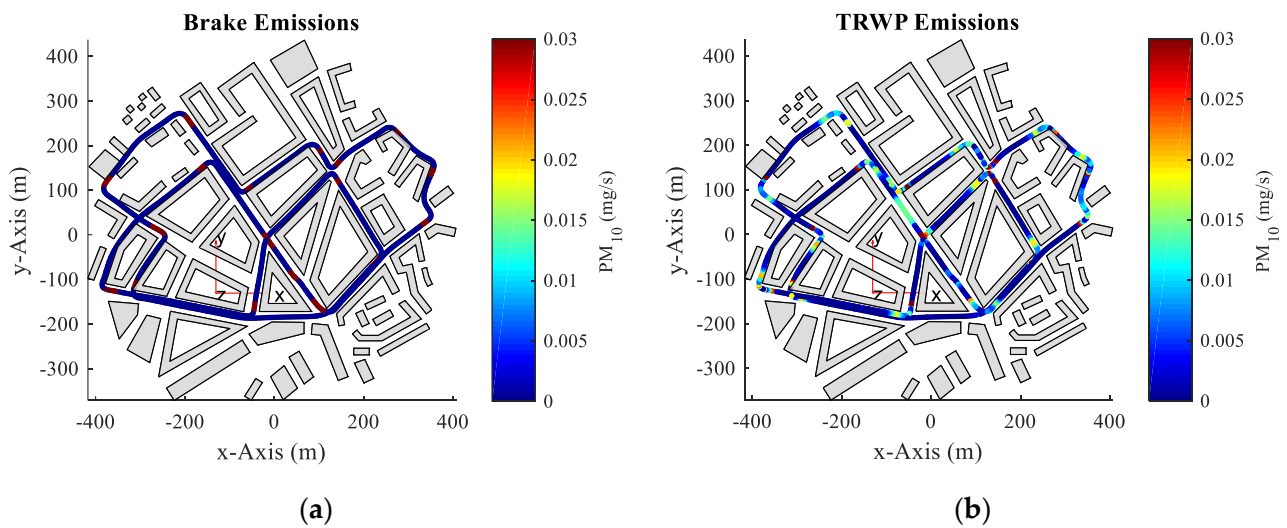


Figure 7. Cont.

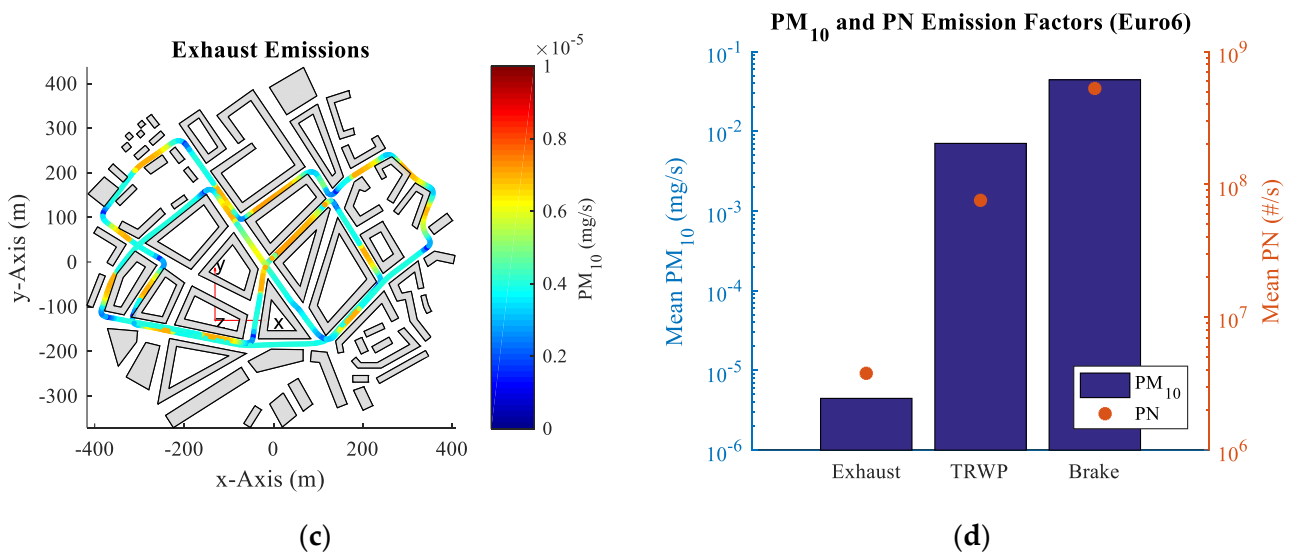


Figure 7. Estimated vehicle-related PM₁₀ contribution within a virtual city environment: (a) brake wear, (b) TRWP, (c) exhaust emissions, (d) virtual PM₁₀ and PN emission factors.

Based on the estimation, brake wear appears to be the biggest source of vehicle-related particle emissions for a vehicle with conventional combustion engine. TRWP is responsible for the second highest contribution, while exhaust particles do not have a significant influence for vehicles with modern exhaust-after-treatment systems. It also has to be taken into account that potential sampling losses for brake and TRWP emissions were not considered. Taking into account the speed profile of the virtual test cycle, it would be possible to calculate emission factors per distance (#/km) and (mg/km). However, it was avoided, since the relationship of brake and TRWP emissions for subject to considerable uncertainties (e.g., in relation to the measurement efficiency). Therefore, it would be risky to compare emission factors with current legal regulation values for exhaust particles.

3.3. Prediction of Particle Dispersion in the Environment

In the next step, the correlation between particle emissions and the ambient wind direction is investigated. Furthermore, local PM₁₀ concentrations are determined. Therefore, the previously predicted emissions are transferred into the flow model described in Figure 3. Furthermore, the particle size distribution (PSD) must be taken into account. The size distribution describes the fraction of particles, which lie within a specific size range relative to total number or mass of the dust sample. Depending on the particle diameter and density, particle trajectories deviate from the streamlines of the flow. Fine particles ($d_p < 1 \mu\text{m}$) tend to follow the air flow and remain in the ambient air for a long period of time. Rather coarse particles ($d_p > 1 \mu\text{m}$) deviate from the streamlines resulting in a higher deposition rate and a shorter residence time. To consider the different aerodynamic properties, the previously predicted particle streams are divided into several sub-streams according to the particle size distribution. Figure 8a shows the number and mass related particle size distribution of a TRWP sample, which was collected on a filter during the test drive. The sample was analyzed based on an optical microscope (magnification: 400 \times). The number of particles and the related particle sizes were determined using an image recognition software. According to the mass related size distribution, the previously predicted TRWP particle stream was subdivided into ten discrete particle streams (size range: 1 μm –10 μm). The particle streams were injected into the flow field to determine the local PM₁₀ concentration as a result of the TRWP contribution.

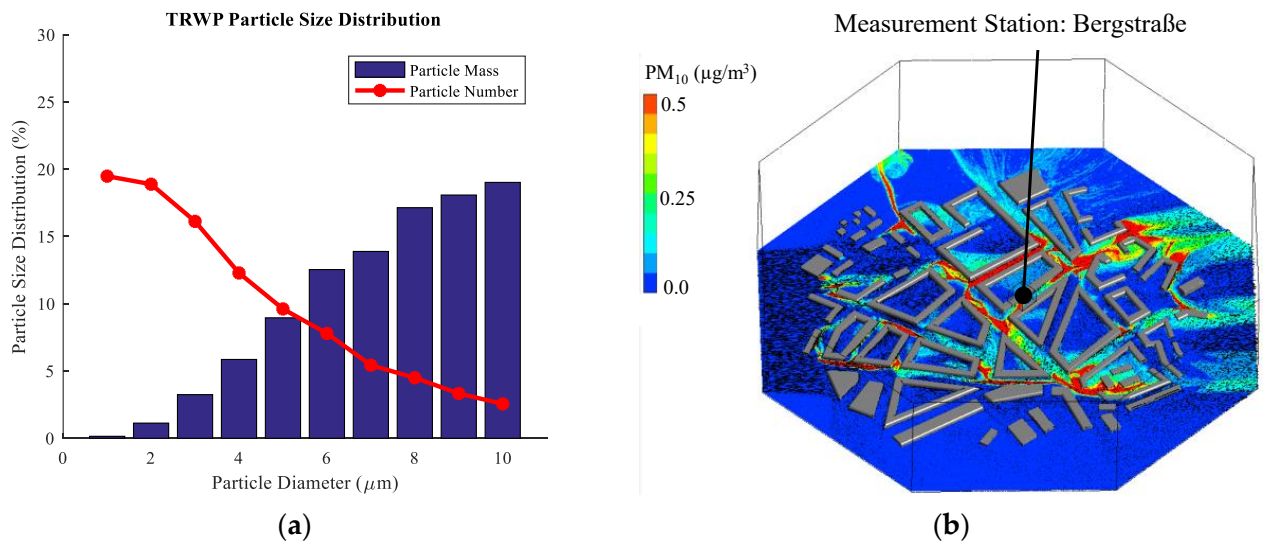


Figure 8. Transfer of a predicted particle emission stream into the flow model considering the particle size distribution: (a) particle size distribution of a TRWP sample; (b) dispersion of TRWP particles within the road canyons of the city model (global wind direction: west/global wind speed: 10 km/h).

The flow model is used to estimate the particle concentration within a specific area and for defined wind conditions. The subject of further investigation is a control volume ($40 \times 40 \times 40 \text{ m}^3$) placed in the street “Bergstraße”, where the environmental measurement station is located. Furthermore, the effect of the global wind speed and the global wind direction was investigated. The dispersion of the particles within the urban area was analyzed for different wind speeds ranging from 5 to 35 km/h and eight global wind directions. A wind speed of 5 km/h corresponds to the minimum (18%) to the average (25%) and a wind speed of 5 km/h corresponds to the minimum (18%) to the average (25%) and 35 km/h to the maximum wind speed measured during the year 2022. West is the most common global wind direction in the city of Erfurt (23% in 2022). Furthermore, six different measurement heights were investigated ranging from 1 to 32 m above the road surface. The height of the aerosol inlet of an environmental measurement station according to 2008/50/EG should be 1–4 m above the ground level. Figure 9 shows the ambient PM_{10} concentration (Bergstraße) as a result of the predicted TRWP particle mass flow. The variations in the global wind speed, the wind direction, and the measurement height lead to major PM_{10} concentration differences. As expected, the highest concentration values occur near the road surface. The concentration level decreases with increasing distance. A higher global wind speed, however, does not necessarily lead to lower PM_{10} values due to increased dilution. This is attributed to the fact that the local wind speed in the street canyon can deviate from the global wind. Furthermore, depending on the wind direction, particles can be trans-located from high emission areas and influence the local PM_{10} concentration elsewhere. The results show that the correlation between vehicle-related particle emissions and the PM_{10} concentration in the ambient air is highly complex and highly influenced by wind conditions and the location of the measurement inlet.

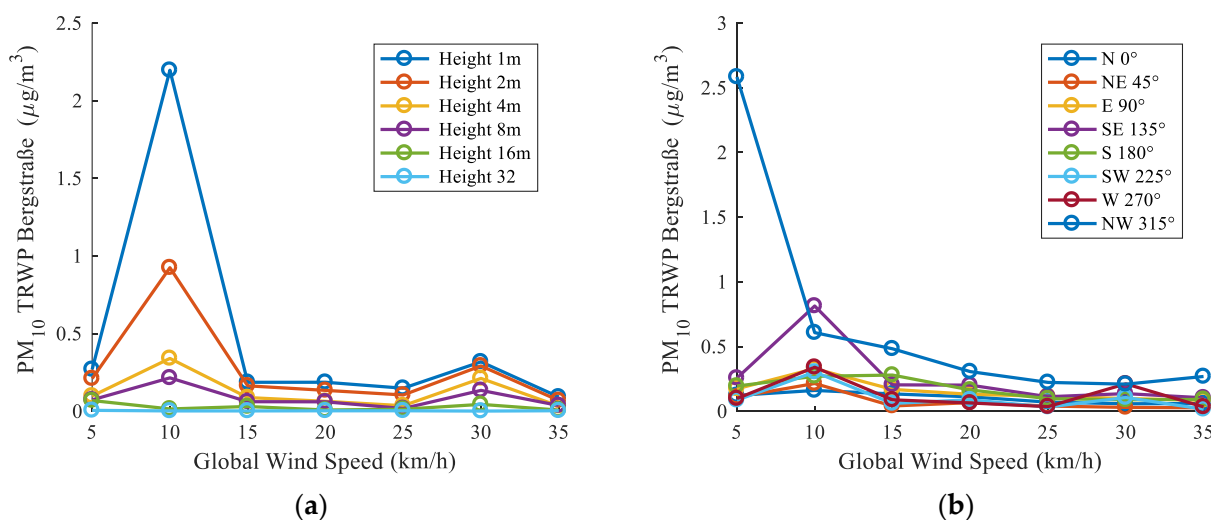


Figure 9. Correlation between global wind conditions and local PM₁₀ concentrations (Bergstraße): (a) influence of global wind speed (global wind direction: west); (b) influence of the global wind direction (height above the road surface: 4 m).

4. Discussion

The aim of this work was to analyze the correlation between vehicle-related particle emissions and the ambient PM₁₀ concentration. In a first step, particle measurement was conducted based on a test vehicle to create an emission data base. Therefore, time resolved particle measurements (exhaust, brake wear, TRWP) as well as vehicle parameters were recorded. In a next step, the major influence parameters were identified. The major recorded parameters for the major influence parameters were identified. The major influence parameters of particle formation were found to be engine rotational speed and torque for exhaust, particle emission longitudinal vehicle speed and slip velocity for TRWP emissions and lateral and longitudinal velocity for the prediction of particle emission. Consequently, a methodology for the prediction of particle emissions based on power transmission was derived. Even though this methodology has a simplified character and does not cover all relevant influence factors, it can be applied to estimate and compare vehicle-related emission sources qualitatively. The models need to be refined in the future based on further parameters (i.e., tire temperature, disc temperature, material properties, etc.).

In the next step, a virtual city model was set up within a vehicle simulation environment. The emission predictors for all three vehicle-related particle sources were implemented to investigate the local particle contribution of each source. It was found that for modern vehicles (fully electric) with emission on combustion engines, brake wear and TRWP emissions are the adaptation of vehicle speed and driving direction. To investigate the influence on the ambient PM₁₀ concentration, a corresponding CFD-based flow model of the city environment was set up as well. Furthermore, a methodology was developed to transfer the predicted emissions into the flow model, while considering the aerodynamic properties of the particles (particle size distribution). Finally, the resulting ambient PM₁₀ concentration was estimated as a function of the global wind speed, wind direction and the filter height. It was found that the highest ambient concentrations occur close to the road surface and show a decreasing tendency with increasing distance. Higher wind speeds wind direction and the filter height. It was found that the highest ambient concentration effect of particle formation and global wind speed increase with increasing distance as a function of wind speed and direction. In general, it showed that ambient PM₁₀ concentration levels vary greatly due to wind conditions even when the same particle contribution is applied.

The presented method allows the estimation of the generation of particle emissions from different sources depending on the vehicle type and driving behavior. Furthermore, the influence on the ambient air quality can be estimated taking into account global and

The presented method allows the estimation of the generation of particle emissions from different sources depending on the vehicle type and driving behavior. Furthermore, the influence on the ambient air quality can be estimated taking into account global and

the influence on the ambient air quality can be estimated taking into account global and local wind conditions as well as particle transport within the street canyons (heat map). Therefore, the presented approach represents an excellent tool for the assessment of the effectiveness of protective measures (i.e., speed limits, bans for certain vehicle types, etc.).

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Abbreviations

List of abbreviations

Abbreviation	Name
PN	Particle Number
PM	Particulate Matter
TRWP	Tire and Road Wear Particles
CFD	Computational Fluid Dynamics
ANN	Artificial Neural Network
UFP	Ultra Fine Particles
EFM	Exhaust Flow Meter
CVS	Constant Volume Sampling
DPM	Discrete Phase Model
PSD	Particle Size Distribution
RDE	Real Driving Emissions

List of nomenclature

Symbol	Quantity	Unit
\dot{P}_C	Particle flow (number/mass)	(#/s) (mg/s)
\dot{V}	Sample volume flow	(m ³ /s)
ME	Measurement efficiency	(%)
d_p	Particle diameter	(m)
ρ_p	Particle density	(kg/m ³)
P_b	Frictional power (brake)	(W)
P_t	Frictional power (tire)	(W)
P_e	Engine power	(W)
p_b	Brake pressure	(bar)
A_b	Brake pad contact area	(m ²)
μ_b	Coefficient of friction (disc/pad)	(-)
v_b	Brake disc velocity	(m/s)
m_v	Vehicle mass	(kg)
a_{xy}	Longitudinal/lateral acceleration	(g)
ρ_a	Density (air)	(kg/m ³)
c_d	Drag coefficient	(-)
A_v	Cross sectional area (vehicle)	(m ²)
v_v	Vehicle speed	(m/s)
v_s	Slip velocity	(m/s)
n_e	Engine rpm	(1/s)
M_e	Engine torque	(Nm)

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